Advanced Modeling of Transient Effects in Fluid Circuits and Engines

Christian Rathberger, ECS, Magna Powertrain



Where it all comes together."

METALFORMING • ENGINEERING & ASSEMBLY • SEATING • VISION • POWERTRAIN • CLOSURES • EXTERIORS • INTERIORS • ELECTRONICS • ROOF SYSTEMS

Agenda

Introduction

- "KULI transient 2.0"
- Transient delays
- Engine model
- Conclusions



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02.03.2007

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From our customer feedback we see that ...

- ... the **stationary behavior** of cooling systems becomes better and better understood.
- ... people generally have gained a lot of **experience** with simulating cooling systems,
- ... there is a clear trend towards transient simulation.

Therefore one of our key goals is to **boost the transient** capabilities of KULI.



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Improvements in the field of transient simulation include:

• "KULI transient 2.0": KULI 8

- Variable transient step width
- Decoupling of simulation and evaluation step width
- Improved stability and speed
- Flow length dependent delay of components and transient diffusion of thermal shocks. KULI 8
- Improved transient behavior of KULI engine models (e.g. hot soaking) KULI 7.0 / 7.1



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Diploma thesis and cooperation with the Institute for **Numerical Mathematics** at the **Johannes Kepler University Linz**



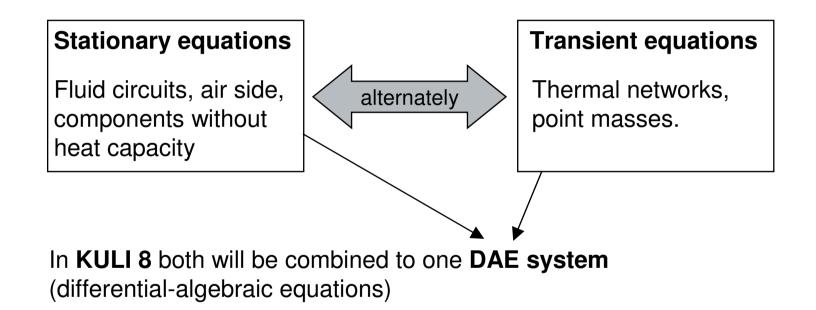
- Thorough analysis of transient simulation in KULI 7
- Test of several different transient solver strategies
- Tailor made implementation based on solid mathematical background
- Test and comparison with current solver



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Up to now stationary and transient parts of the cooling system were solved alternately:





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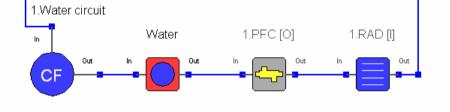
Reference example

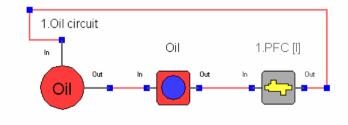
For the analysis we use a very simple example:

Heat **sources**

Heat **sinks**

We compare the **temperatures** of the point masses.

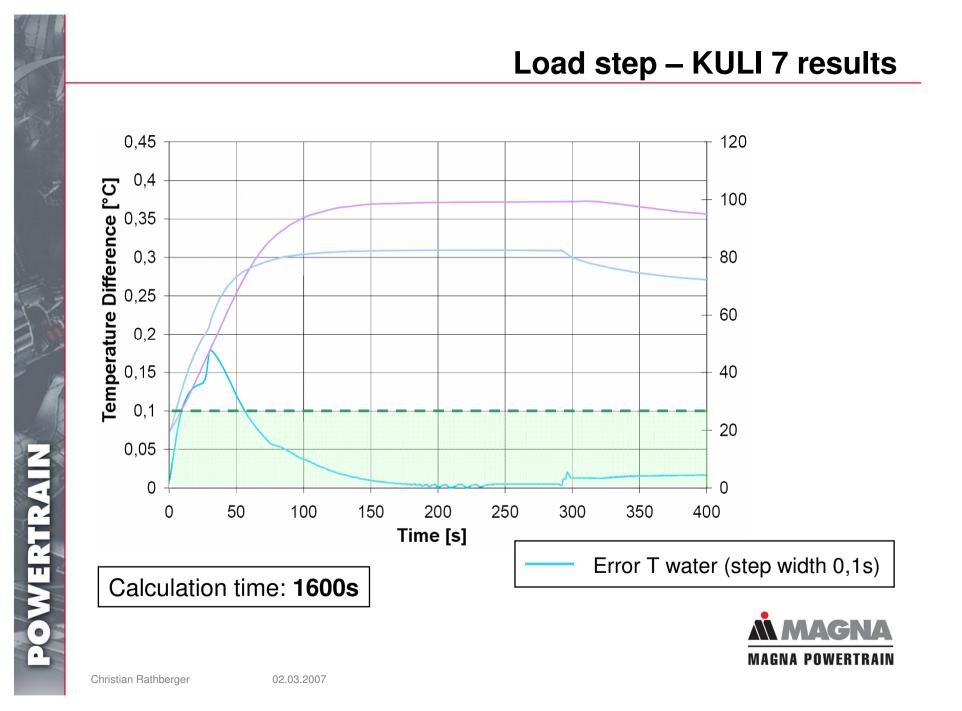






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The key advantages of the new strategy are:

- Much faster (factor 10 for test case)
- More **user-friendly** (user defined step width changes only output density, but not the results)
- Automatic **adaptive step width** allows user defined solver precision. Algorithm detects critical points.





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If we take a tube with

Tube length: 75cm Diameter: 15mm

and assume

Mass flow: 0,5kg/s

we get a flow speed of ~0,7m/s and thus a transient delay of ~1s.

An abrupt temperature change takes **more than one second** to reach the other end of the tube.

Delays can have an effect on transient simulations!



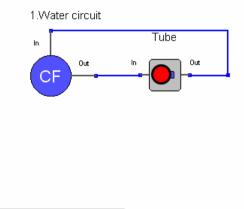
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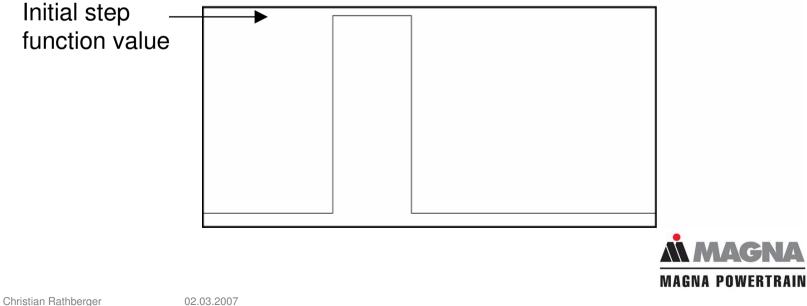
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If we model the delay in a tube and apply a **temperature step function** to a circuit...

... this step circulates in the System "for ever".

This obviously is not what happens in a **real** cooling system!

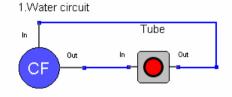


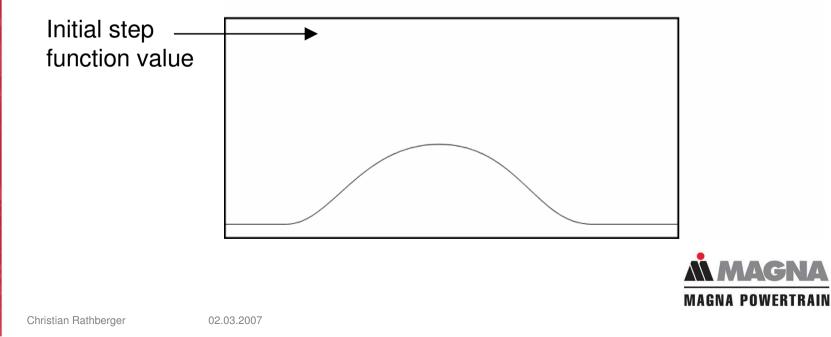


In a real cooling system the temperature step function will **diverge** as time passes.

It will flatten more and more... this can be modeled by **diffusion** processes.

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Hot

In a tube the flow speed varies across the diameter.

In the center the flow speed is always highest (e.g. **Hagen-Poisseuille** for laminar flows)

A heat step will reach the end of a tube in the center first.

The speed profile depends on the **Reynolds number**.

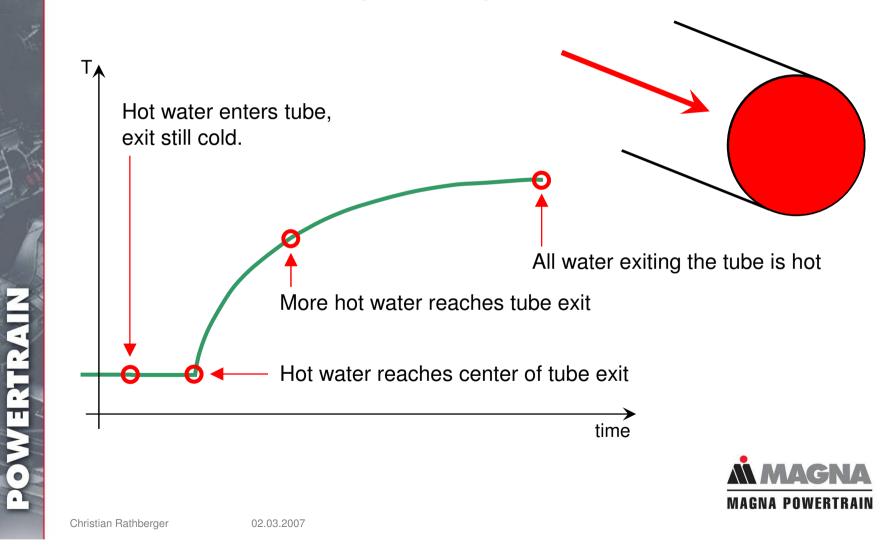


Cold

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We now look at the average exit temperature of a tube:



To calculate this effect in KULI, no additional measurement data is required.

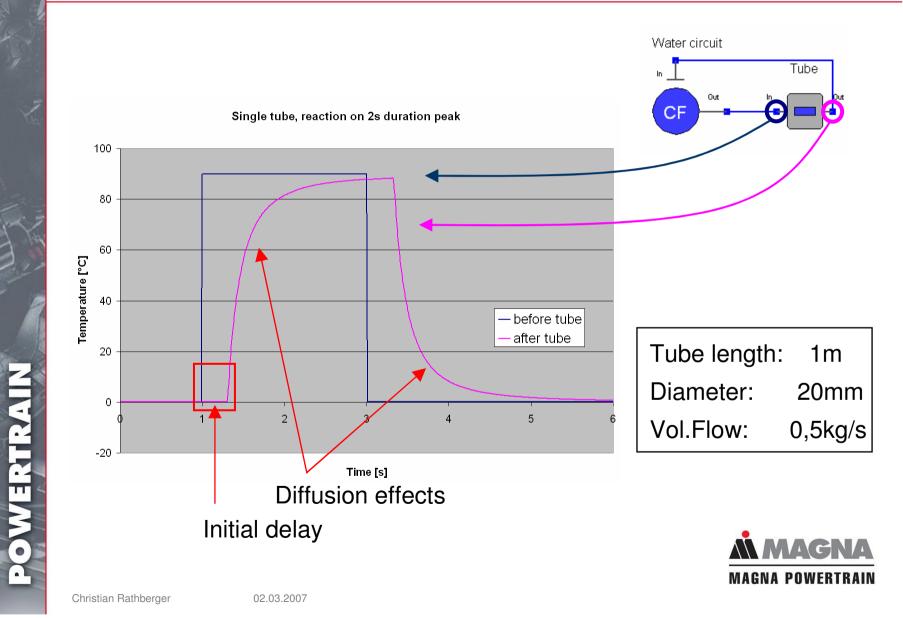
Component parameters **Inner diameter** ID: 1.TUB Tube Comments and tube length 10 Inner diameter (mm) Calc, method for pressure drop Standard 0.06 Pipe roughness [mm] C Miller 2000 Tube length [mm] are already available! 🖄 🤌 🖪 Ok Component Tube_L2m_D10mm.tub Cancel 1.TUB

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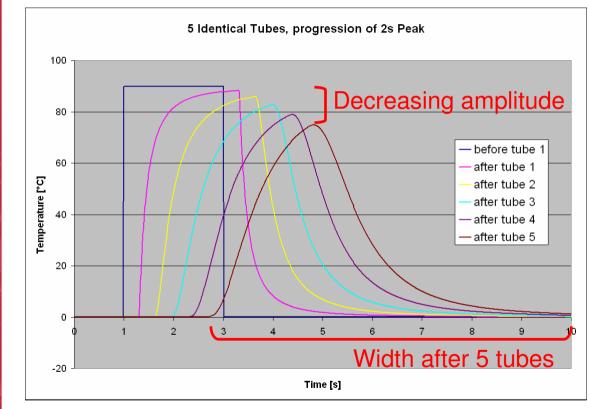
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Transient delay and diffusion of a single tube





Delay and diffusion, 5 consecutive tubes

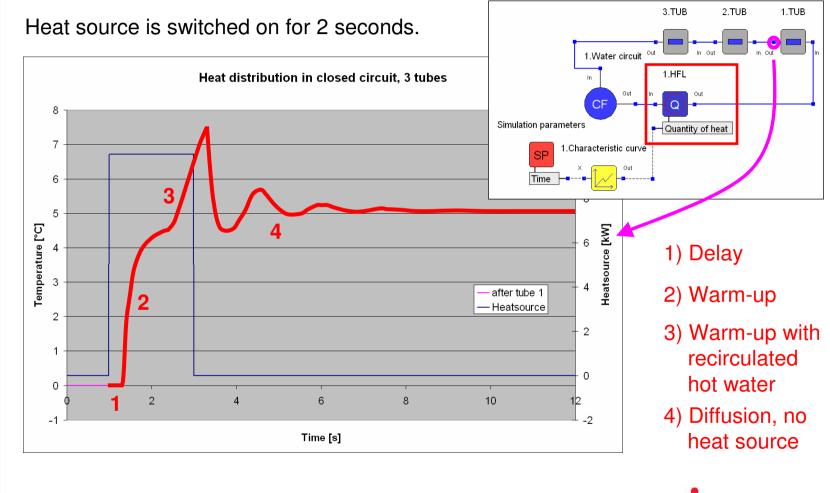


Same situation as before, this time 5 consecutive tubes.



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Closed Circuit





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- Different flow speeds different speed profiles
- No 3D modeling, KULI is and remains a 1D tool
- Thermal capacity of media in tubes is considered
- Thermal capacity and heat conduction to tube walls modeled by point masses.
- This approach also improves handling of zero mass flow operating points



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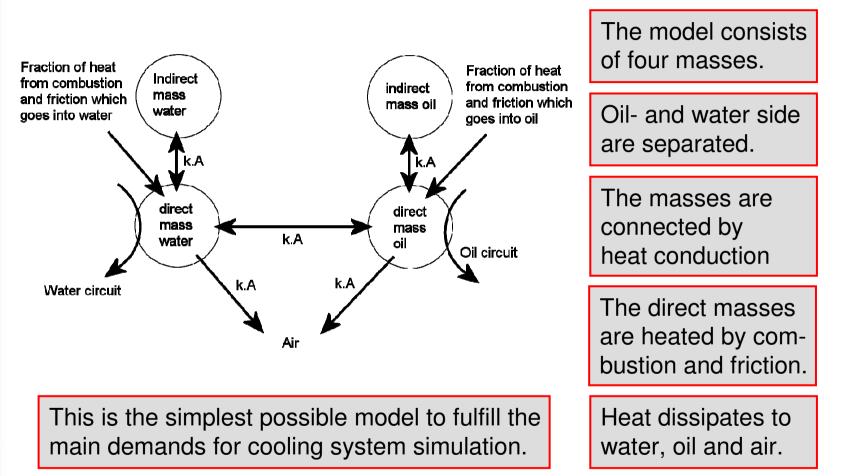
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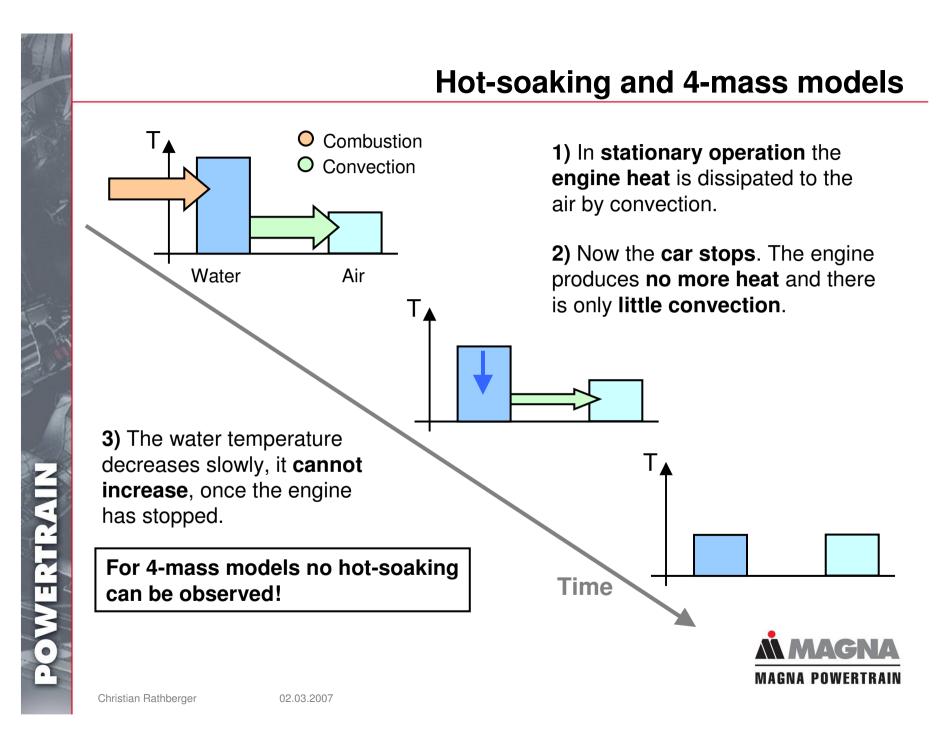
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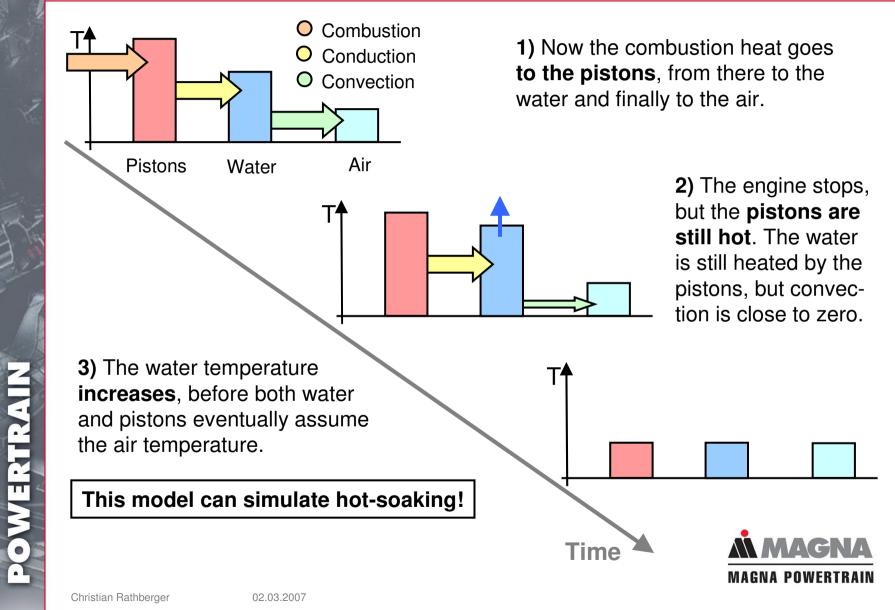


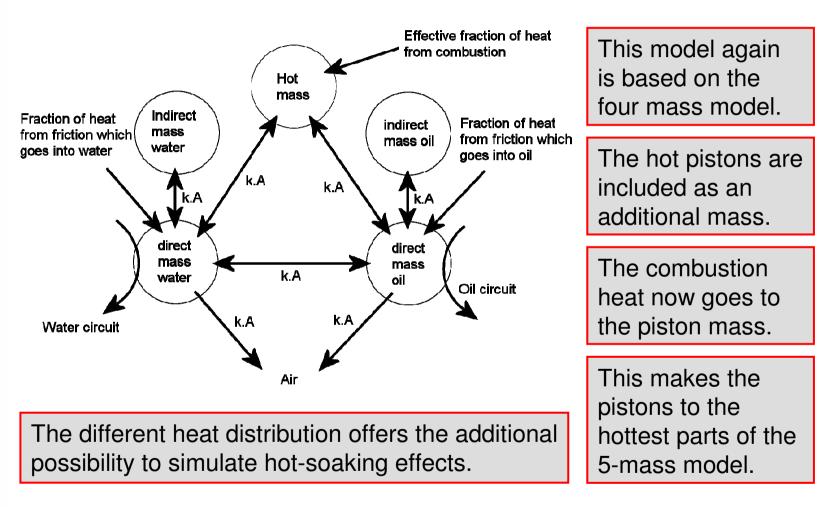


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Hot-soaking and 5-mass models



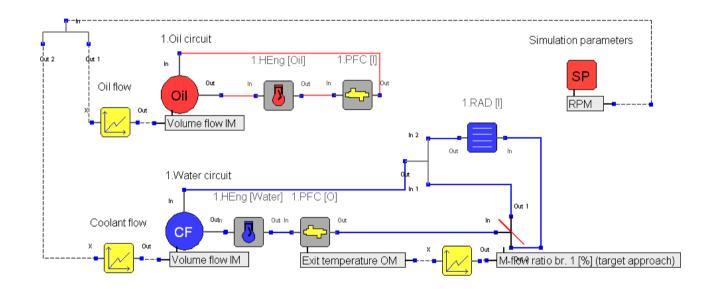




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We use a **5-mass model** for a truck engine and a simplified cooling system model to first **heat up the engine** at a high operating point and then **stop abruptly**.



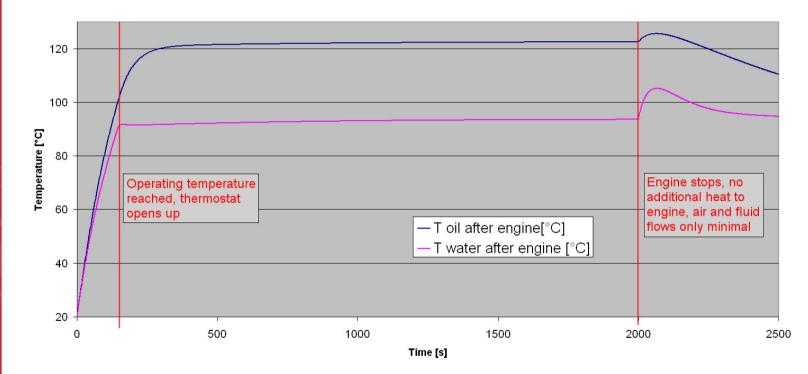


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After high load stationary operation the vehicle stops...



... the Cooling air flow stops, but the engine is still heating up the water and oil circuits.

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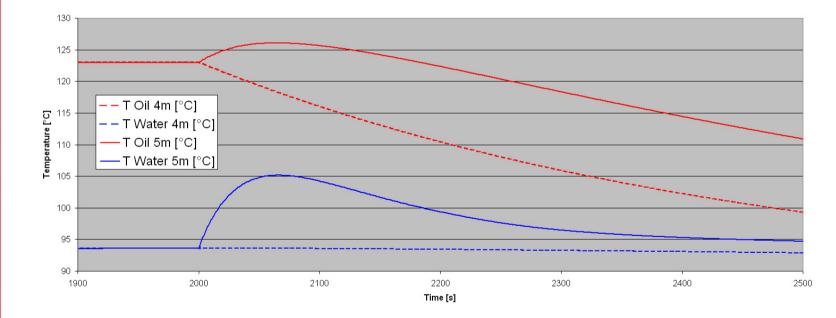
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Comparison of 4- and 5-mass engine models

For 4-mass engine models no hot soaking can be simulated.





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Simulation of **complex transient effects** requires relatively high

temporal resolution (transient solver) and

geometrical resolution (tube, engine-model).

Time steps should be kept short **only where necessary**. **Adaptive time discretization**.

3D discretization has to be **avoided** (calculation time!). Good **approximations** and **closed mathematical formulations** should be used whenever possible!



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Discussion...

